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MADE EASY CIVIL ENGINEERING Strength of Material BY-Rishi Sir

- Theory
- Explanation
- Derivation
- Example
- Shortcuts
- Previous Years Question With Solution

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- GATE } FY
- ESE }
- IFS or
IAS
- Last 5 yr

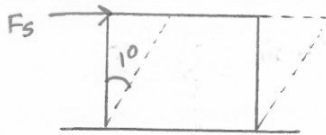
STRENGTH OF MATERIAL

~ Rishi Sir ~
{ 01/09/25 }

1.1)

Engineering Sciences.

Solid Mechanics



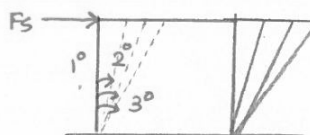
$$t = 1 \text{ sec} \Rightarrow 1^\circ$$

$$t = 2 \text{ sec} \Rightarrow 1^\circ$$

$$t = 3 \text{ sec} \Rightarrow 1^\circ$$

$$t = 10 \text{ sec} \Rightarrow 1^\circ$$

Fluid Mechanics



$$t = 1 \text{ sec} = 1^\circ$$

$$t = 10 \text{ sec} = 2^\circ$$

$$t = 20 \text{ sec} = 3^\circ$$

Properties of Material.

Engineering Mechanics

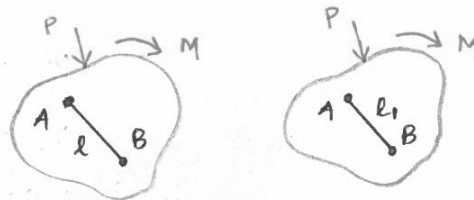
{ Study of Rigid Body }



Strength of

Material: / SOM / MOS / MODB

{ Study of Deformable Bodies }



$$\bullet \text{ Strain} = \frac{\text{Change in length}}{\text{Initial Length}} = \frac{L_f - L_i}{L_i} = \frac{l_1 - l}{l}$$

Deformation.

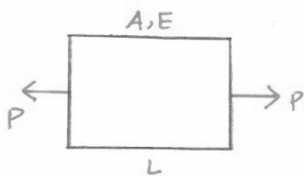
Elastic (✓)

Plastic (x)

Mild Steel $\Rightarrow$ Universal Testing Machine.

- A = Proportional Limit Point
- B = Elastic Limit Point
- G = Fracture Point.

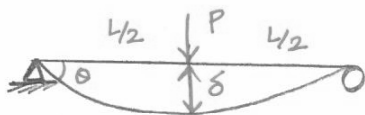
Point (X)



$$S = \frac{PL}{AE} = \text{SOM (X)}$$

Hoffman's Eqⁿ is used beyond Elastic Limit $[\sigma_F = K \epsilon^m \epsilon^n]$

Plastic Strain $\gg \gg$ Elastic Strain

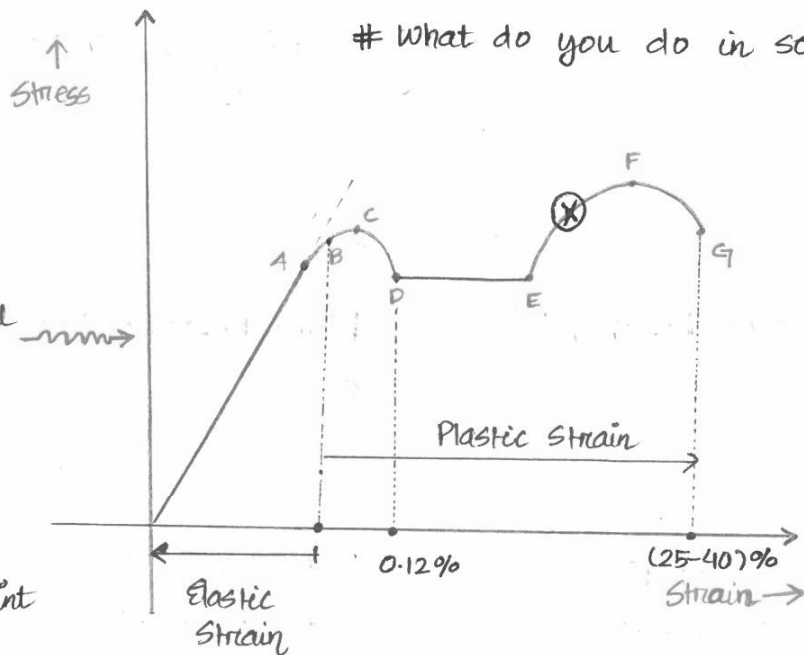


$\theta, \delta \rightarrow$ very small.

$$\delta = \frac{PL^3}{48EI}$$

$$\left\{ \frac{M}{I} = \frac{\sigma}{y} = \frac{E}{R} \right\} \Rightarrow M = EI \frac{d^2y}{dx^2}$$

What do you do in SOM?



$\Rightarrow$ Mathematically A, B, C, D can be taken for calculation as a single point as strain is very less [Hooke's Law]

$\Rightarrow$



Technically Hooke's Law is valid till A.

Differential Equation for R.

$$\frac{1}{R} = \frac{d^2y/dx^2}{\left\{ 1 + \left(\frac{dy}{dx} \right)^2 \right\}^{3/2}}$$

Curvature for small deformation.

for Rubbers

θ', δ'
 $\downarrow$

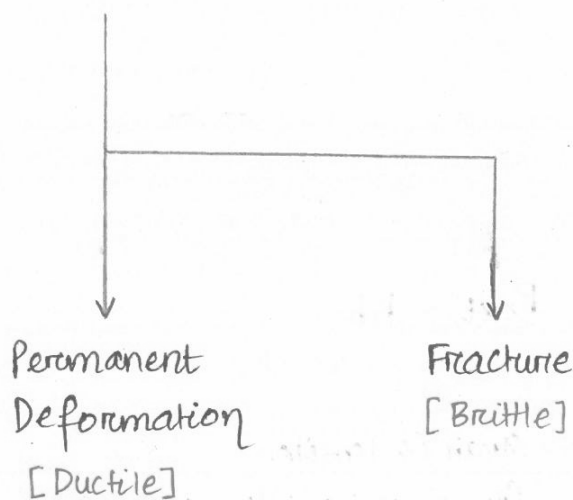
very large.

$$\text{So, } \delta' \neq \frac{PL^3}{48EI}$$

$$\frac{1}{R} \neq \frac{d^2y}{dx^2}$$

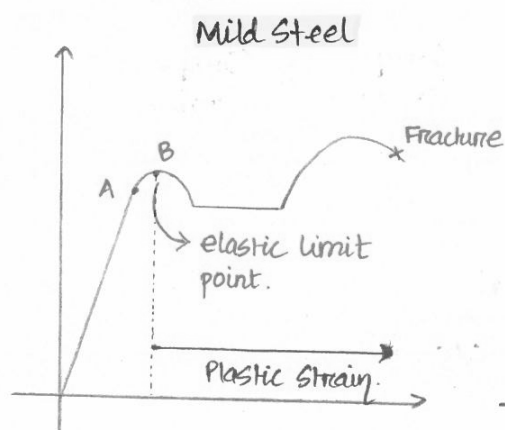
$\Downarrow$
FBD.

Failure of Material

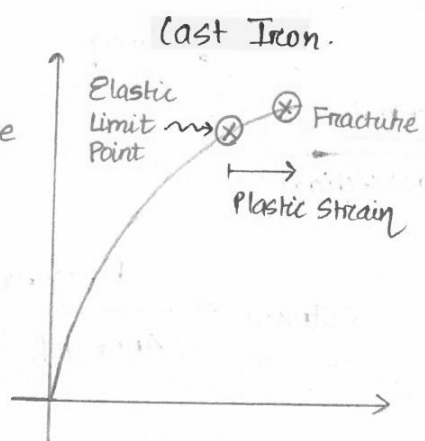


- Longer the x-axis greater is its Ductile Property.

⇒ Greater is its Plastic Deformation.

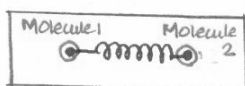


⇒ $\sigma_B = 250 \text{ MPa}$



⇒ Dist. is very less b/w points hence can be treated as a st. line or curve.

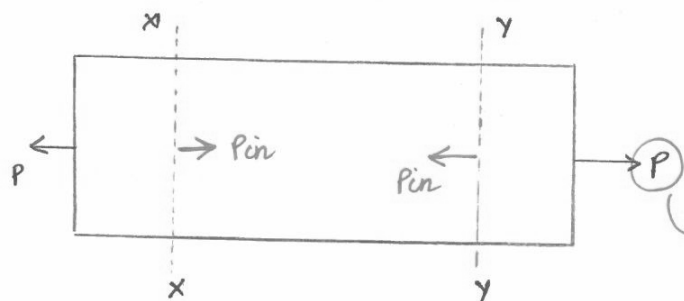
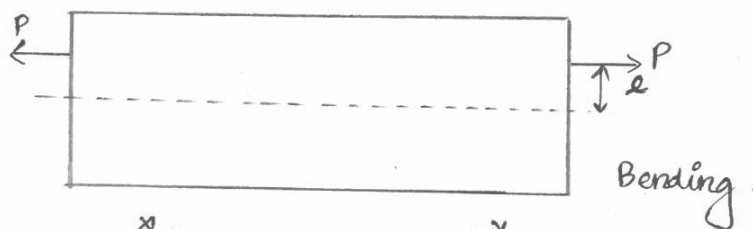
* Difference b/w strength and stress.



Analogy: consider a spring b/w molecules.

- Internal Resisting force against deformation is called stress.
- Stress at which body deforms permanently is its strength.

- ⇒ Strain is a fundamental unit.
- Stress is a cause of strain.
- the stress at which body deforms permanently is called strength of material.



$P_{ext} = P_{in}$ { bound to happen else body will accelerate }

↳ Strain is tensile.

for σ we have to cut the c/s.

$\sigma \Rightarrow$ tensile

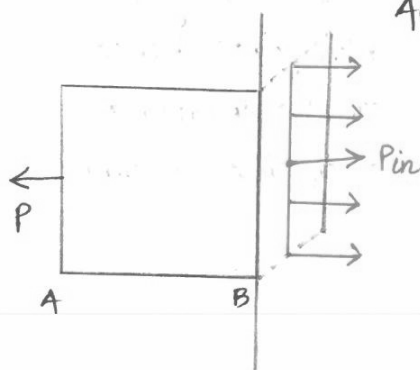
$[L + \Delta L]$

$$\epsilon = \text{Strain} = \frac{\Delta L}{L} = \frac{(L + \Delta L) - L}{L}$$

Hooke's law

$\sigma \propto \epsilon$

$$\sigma = E\epsilon \Rightarrow \sigma = \frac{P_{\text{internal}}}{\text{Area of cross section}}$$



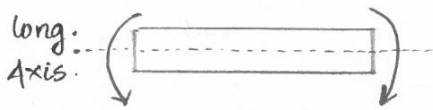
$$\sigma_{\text{stress}} = \frac{P_{\text{internal}}}{\text{Area of c/s.}}$$

- Tensile Force $\Rightarrow$ Tensile Stress $\Rightarrow$ Tensile Strain { visualise by cutting }
- Comp. Force $\Rightarrow$ Comp. Stress $\Rightarrow$ Comp. Strain

- Strain plotted in x-axis $\Rightarrow$ Hence a fundamental unit
- Strain is cause of stress
- Strain is fundamental unit & stress is derived from.
Strain
- Stress is internal resisting force offered by material
against deformation.

Shear Force And Bending Moment Diagram.

- Diff. b/w bending and twisting?



Bending $\Rightarrow$ Transverse Moment.



Twisting $\Rightarrow$ longitudinal Moment.

[Right Hand Thumb Rule]

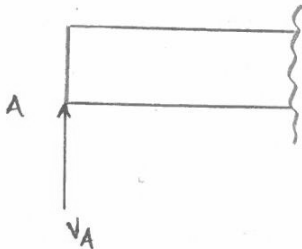
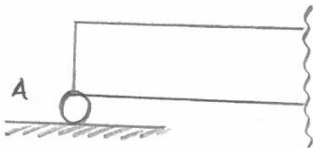
- SFD, BMD.



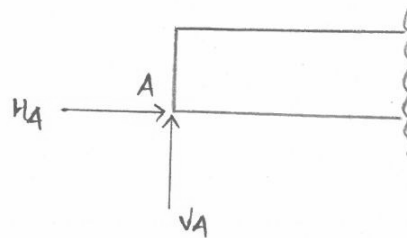
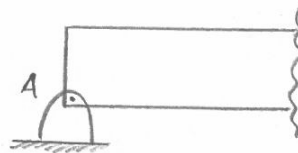
- Analysis of Critical Points/Locations to ensure safety and security of structure against failure.
- If critical points are safe rest points are also safe.

Types of Support

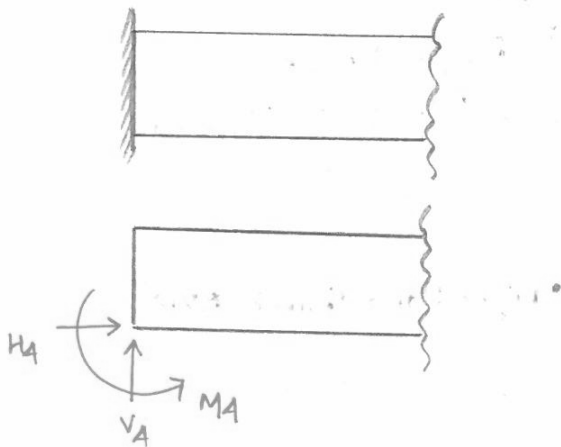
a) Roller Support / Free Support.



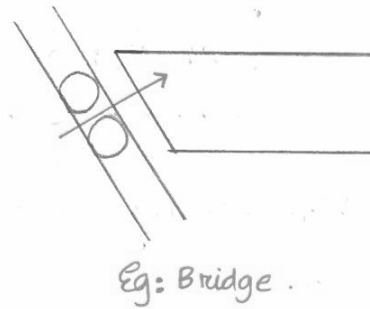
b) Hinge Support / Pinned



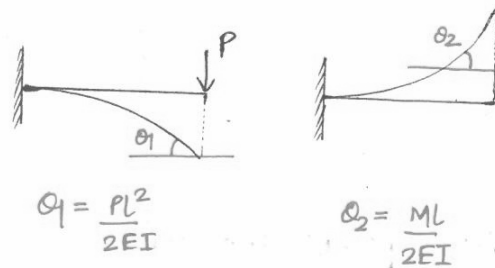
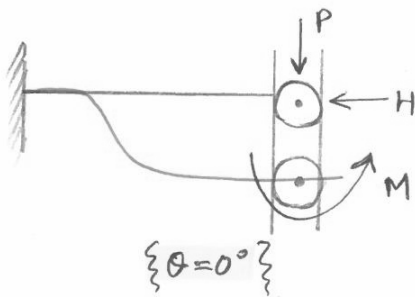
c) fixed Support Reaction.



d) Inclined Roller Support



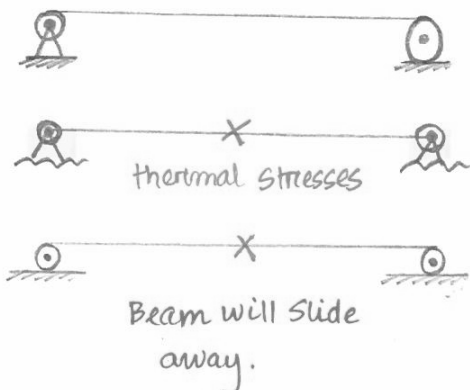
e) Guided Roller Support



Types of Beam.

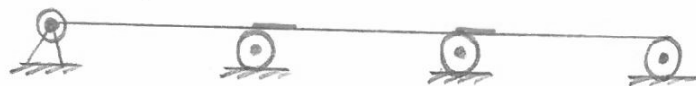
Simply Supported Beam.

{ for short span }
{ allows rotation }

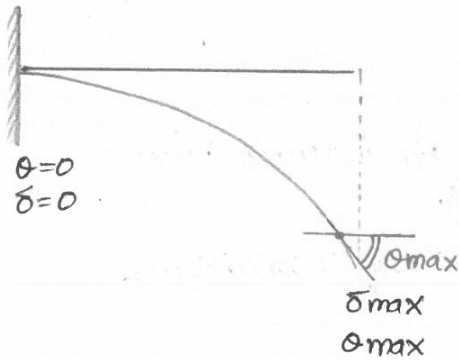


Continuous Beam

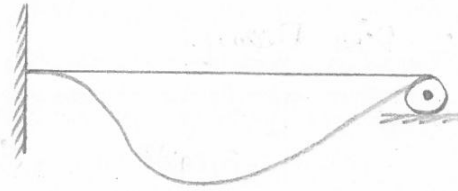
{ for long span }



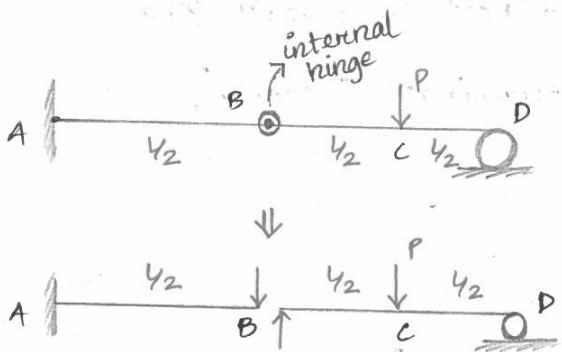
Cantilever Beam



Propped Cantilever Beam



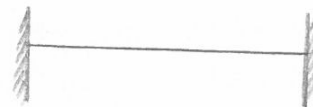
Beam with internal Hinge



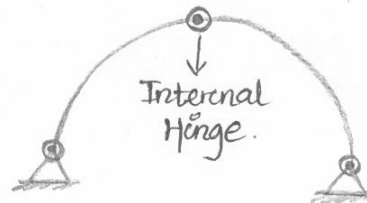
$$(B \cdot M)_{\text{internal hinge}} = 0$$

- Moment of left section is not transferred to right section. and vice versa.

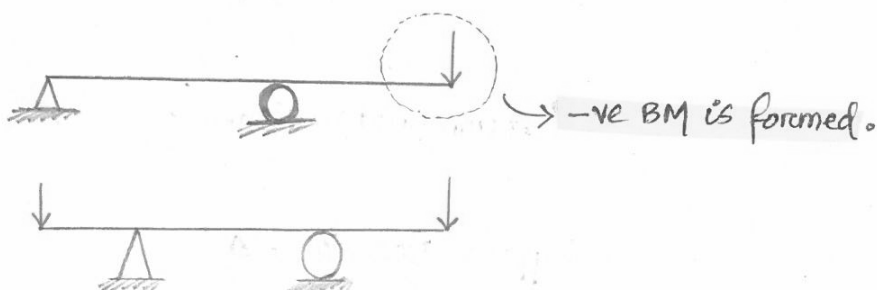
Fixed Beam



- Most commonly used in structure.



Overhanging Beam



Analysis of Beam.

02/09/25

- for 2-Dim Beam.

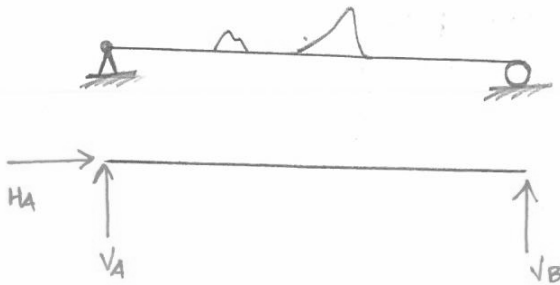
$$\left. \begin{array}{l} a) \sum F_x = 0 \\ b) \sum F_y = 0 \\ c) \sum M_z = 0 \end{array} \right\} \begin{array}{l} 3 \text{ eqbm} \\ \text{equation} \end{array}$$

if beam has an internal hinge

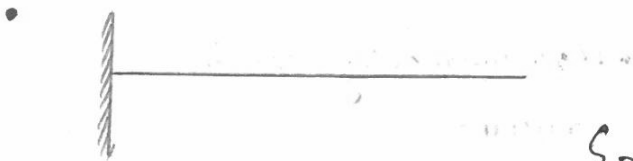


1 additional eqbm condition.

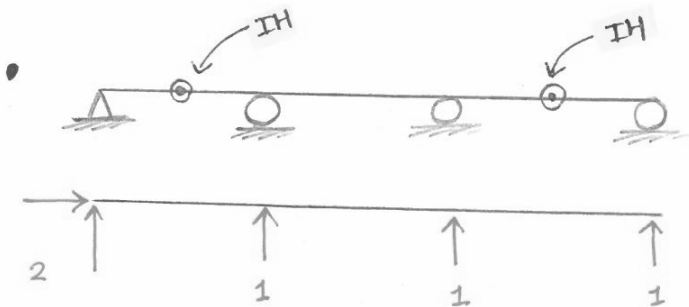
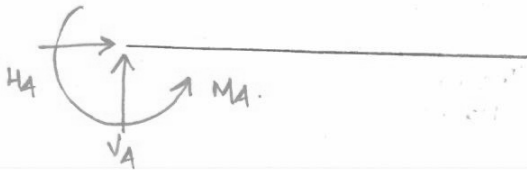
- Eg:



3 Support Reactions $\Rightarrow$ 3 eqbm condⁿ
 {Determinate Beam.}



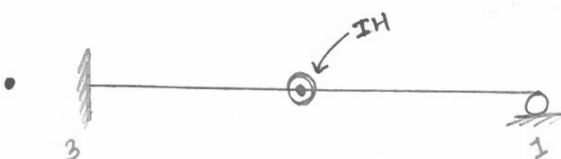
{Determinate Beam.}



$$\text{Support Reaction} = 2 + 1 + 1 + 1 = 5$$

$$\text{Eqbm Equation} = 3 + 2 = 5$$

{Determinate Beam}



{Determinate Beam}

$$\text{Support Reaction} = 4$$

$$\text{Eqbm Eq}^n = 3 + 1 = 4$$